

ELE 457 Feedback Control System: System Modeling

Fall 2009

Review of Last Lecture

- System Modeling: Differential Equation
- Complex variable and S-plane
- Laplace Transform
- Theorem of Laplace Transform

Theorem of Laplace Transform

- Linear

$$L[f_1(t)] = F_1(s);$$

$$L[f_2(t)] = F_2(s)$$

$$L[af_1(t) + bf_2(t)] = aF_1(s) + bF_2(s)$$

- Scale

$$L[f(at)] = \frac{1}{a} F\left(\frac{s}{a}\right)$$

Theorem of Laplace Transform

- Frequency shift

$$L[e^{-at} f(t)] = F(s + a)$$

- Shifting

$$L[f(t - t_0)u(t - t_0)] = e^{-t_0 s} F(s)$$

Example: $F(s) = \frac{\omega}{(s + a)^2 + \omega^2}$

Give $L[e^{-t}] = \frac{1}{s+1}$, $L[e^{-t+3}u(t-3)]$? $L[e^{-t+3}] = ?$

what about $L[e^{-(t+3)}]$? $L[e^{-(t+3)}u(t+3)]$?

Theorem of Laplace Transform

- Integral

$$L\left[\int_0^t f(\tau) d\tau\right] = \frac{F(s)}{s}$$

- Convolution

$$L^{-1}[F_1(s)F_2(s)] = \int_0^t f_1(t-\tau)f_2(\tau)d\tau$$

Time-domain convolution becomes the frequency-domain multiplication!!!

Theorem of Laplace Transform

- Derivative

$$L\left[\frac{df(t)}{dt}\right] = sF(s) - f(0^-)$$

If signals $f(t)$ is discontinuous at $t=0$

Time-domain differentiation becomes multiplication by frequency variable $s!!!$

- Nth-order Derivative

$$L\left[\frac{d^n f(t)}{dt^n}\right] = s^n F(s) - s^{n-1} f(0^-) - s^{n-2} f'(0^-) \dots - f^{(n-1)}(0^-)$$

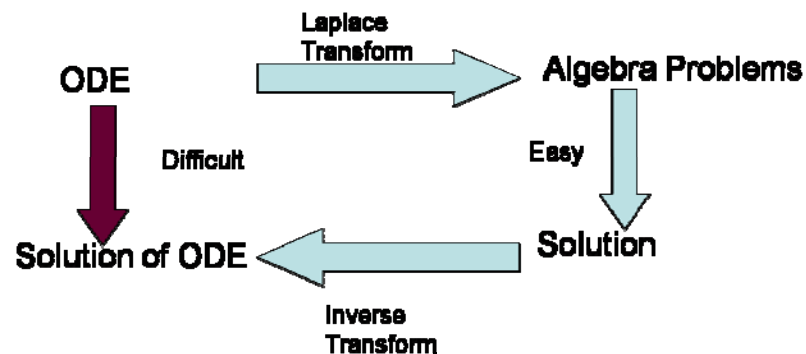
- When finding transfer function, **zero initial condition must be assumed.**
- If the initial value is considered, the initial value can be considered as a impulse input.

Inverse Laplace Transform

- General rational function

$$F(s) = \frac{N(s)}{D(s)} = \frac{b_m s^m + b_{m-1} s^{m-1} + \dots + b_1 s + b_0}{s^n + a_{n-1} s^{n-1} + \dots + a_1 s + a_0} \quad m < n$$

How to obtain the inverse Laplace transform from F(s)?



Inverse LT: Example 1

$$F(s) = \frac{12s + 45}{s^2 + 5s} = \frac{9(s + 5) + 3s}{s(s + 5)} = \frac{9}{s} + \frac{3}{s + 5}$$

$$f(t) = 9 + 3e^{-5t}, t \geq 0$$

$$\text{or } f(t) = [9 + 3e^{-5t}]u(t)$$

Inverse Laplace Transform

- General rational function

$$F(s) = \frac{N(s)}{D(s)} = \frac{b_m s^m + b_{m-1} s^{m-1} + \dots + b_1 s + b_0}{s^n + a_{n-1} s^{n-1} + \dots + a_1 s + a_0} \quad m < n$$

If D(s) does not have repeated roots

$$F(s) = \frac{N(s)}{\prod_{i=1}^n (s - p_i)} = \frac{k_1}{(s - p_1)} + \frac{k_2}{(s - p_2)} + \dots + \frac{k_n}{(s - p_n)}$$

partial fraction expansion

From Residual Theorem ^{$i=0$}

$$k_j = (s - p_j) F(s) \Big|_{s=p_j}$$

$$f(t) = k_1 e^{p_1 t} + k_2 e^{p_2 t} + \dots + k_n e^{p_n t}$$

Inverse Laplace Transform

If $D(s)$ have repeated roots

$$F(s) = \frac{N(s)}{(s-p)^r (s-c)} = \frac{k_{p1}}{(s-p)} + \frac{k_{p2}}{(s-p)^2} + \dots + \frac{k_{pr}}{(s-p)^r} + \frac{k_c}{s-c}$$

$$k_{pi} = \frac{1}{(r-i)!} \frac{d^{r-i}}{ds^{r-i}} [(s-p)^r F(s)] \Big|_{s=p}$$

Example: Inverse LT

$$F(s) = \frac{2s + 3}{s^3 + 2s^2 + s}$$

$$F(s) = \frac{3}{s} + \frac{-3}{s+1} + \frac{-1}{(s+1)^2}$$

$$f(t) = 3 - 3e^{-t} - te^{-t} \quad t \geq 0$$

$$\text{or } f(t) = [3 - 3e^{-t} - te^{-t}]u(t)$$

Example: Inverse LT

$$F(s) = \frac{2s + 1}{s^2 + 2s + 10}$$

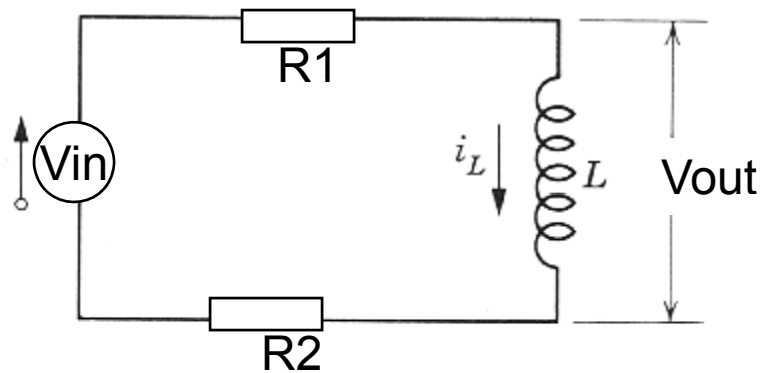
Hint:

$$e^{-at} \sin(\omega t)$$

$$e^{-at} \cos(\omega t)$$

$$\frac{\omega}{(s+a)^2 + \omega^2}$$
$$\frac{s+a}{(s+a)^2 + \omega^2}$$

How to describe a system mathematically?



Kirchhoff's law:

$$V_{in}(t) = (R1 + R2)i(t) + L \frac{di(t)}{dt}$$


$$V_{out}(t) = L \frac{di(t)}{dt}$$

Transfer Function

$$V_{in}(t) = (R1 + R2)i(t) + L \frac{di(t)}{dt}$$

$$V_{out}(t) = L \frac{di(t)}{dt}$$

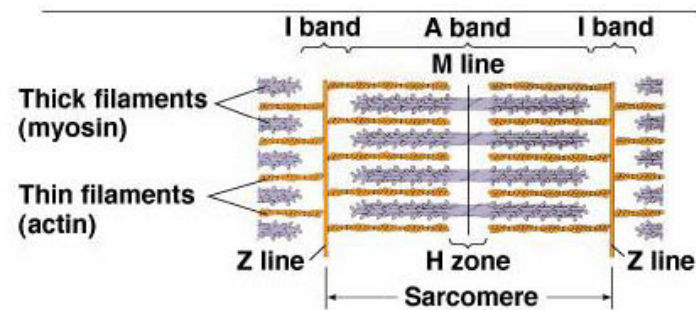
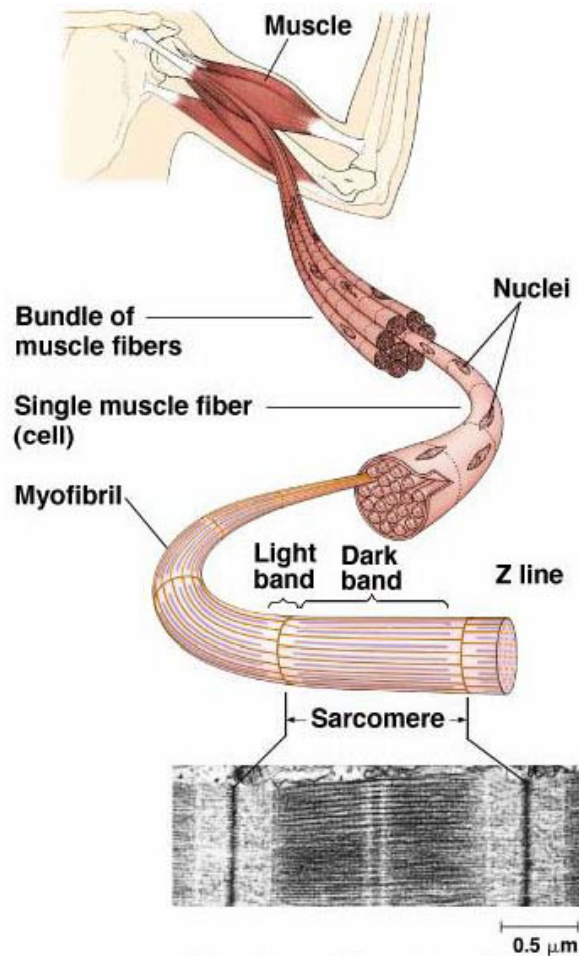
LT:
$$V_{in}(s) = (R1 + R2)I(s) + LSI(s)$$
$$V_{out}(s) = LSI(s)$$

TF: 
$$\therefore \frac{V_{out}(s)}{V_{in}(s)} = \frac{LS}{(R1 + R2 + LS)}$$

A transfer function can be written only for the case in which the system model is a **linear time-invariant** differential equation and the system **initial condition** are ignored.

Mechanical Modeling

- Mechanical System: Skeletal Muscle



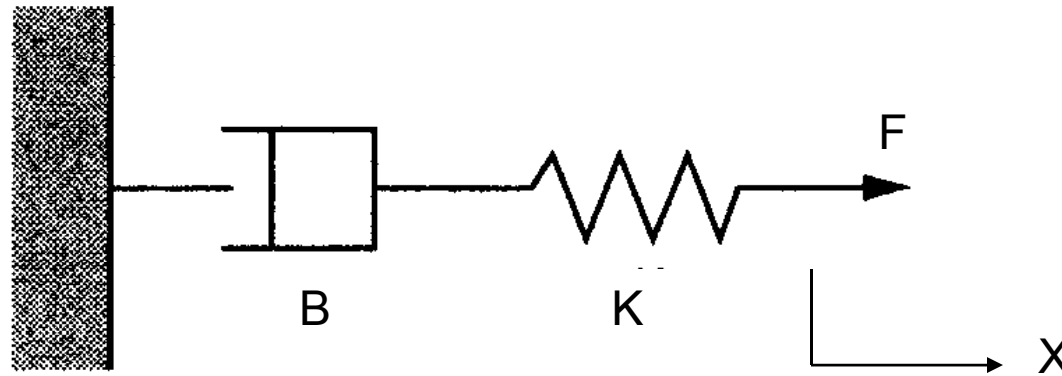
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Viscoelastic Theory

- Generalize materials
- Deformation properties
- With types of elements:
 - Linear elastic elements = springs
 - Viscous elements = dashpots

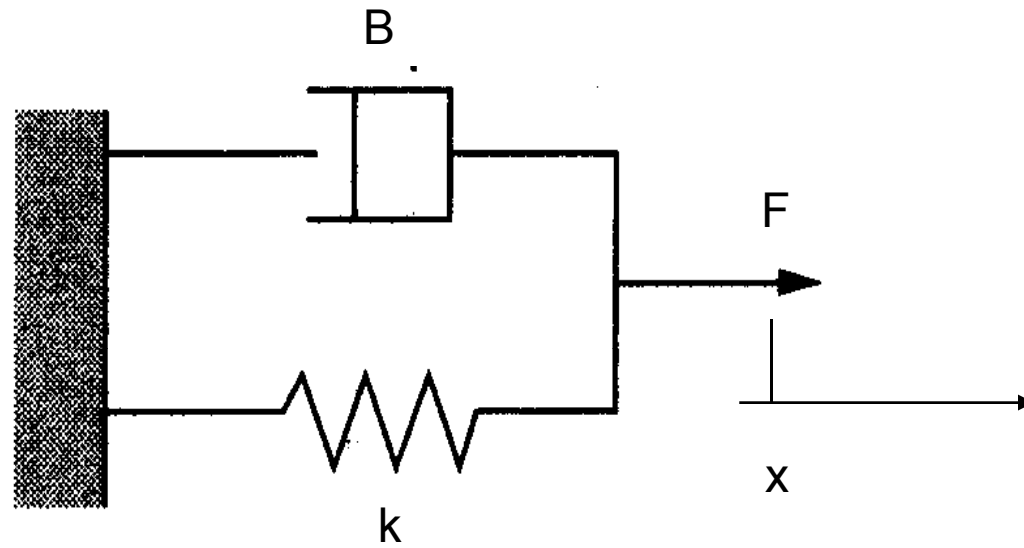
Combinations of Elements #1

Maxwell Model



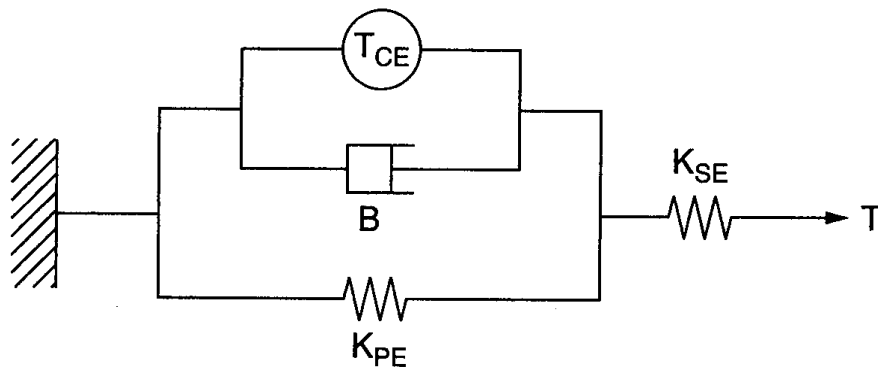
$$\begin{cases} B\dot{x}_1(t) = kx_2(t) = F(t) \\ x_1(t) + x_2(t) = x(t) \end{cases}$$

Combinations of Elements #2



Hill's Muscle Model

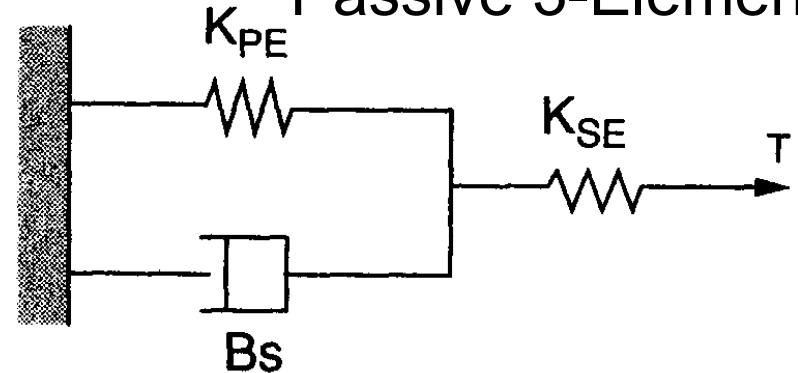
Active 4- element



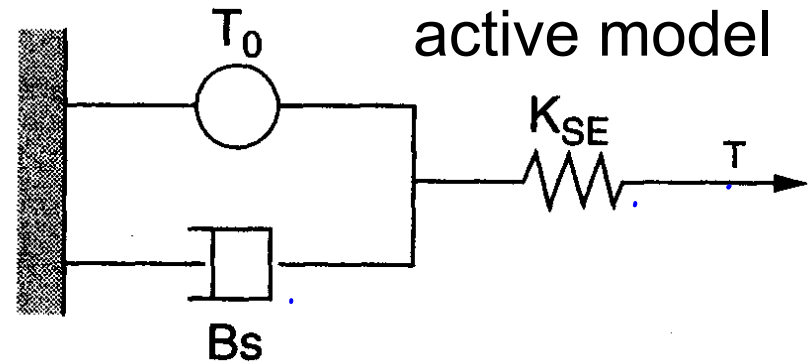
- CE: Contractile element
- SE: Elastic element
- PE: Passive element
- B: Fluid caused damping
- Exertion Force: T-output

Kevin's model

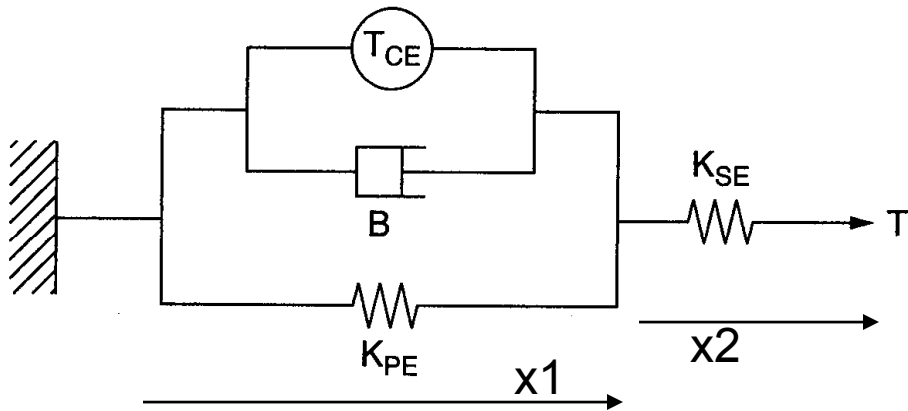
Passive 3-Element



Simplified active model



Hill's Model



$$\left\{ \begin{array}{l} x_1 + x_2 = x \Rightarrow X_1 + X_2 = X \\ T = k_{SE} x_2 \Rightarrow X_2 = \frac{T}{k_{SE}} \\ -T = T_{CE} - B\dot{x}_1 - k_{PE} x_1 \Rightarrow T_{CE} = -T + (Bs + k_{PE})X_1 \end{array} \right.$$

$$T_{CE} = -T + (Bs + k_{PE})(X - X_2)$$

$$T_{CE} = -T + (Bs + k_{PE})\left(X - \frac{T}{k_{SE}}\right)$$

$$T_{CE} = (Bs + k_{PE})X - \left(1 + \frac{Bs + k_{PE}}{k_{SE}}\right)T$$

Hill's Model

$$T_{CE} = (B_S + k_{PE})X - \left(1 + \frac{B_S + k_{PE}}{k_{SE}}\right)T$$

- Discussion

(1) Isometric muscle contraction: $X=0$

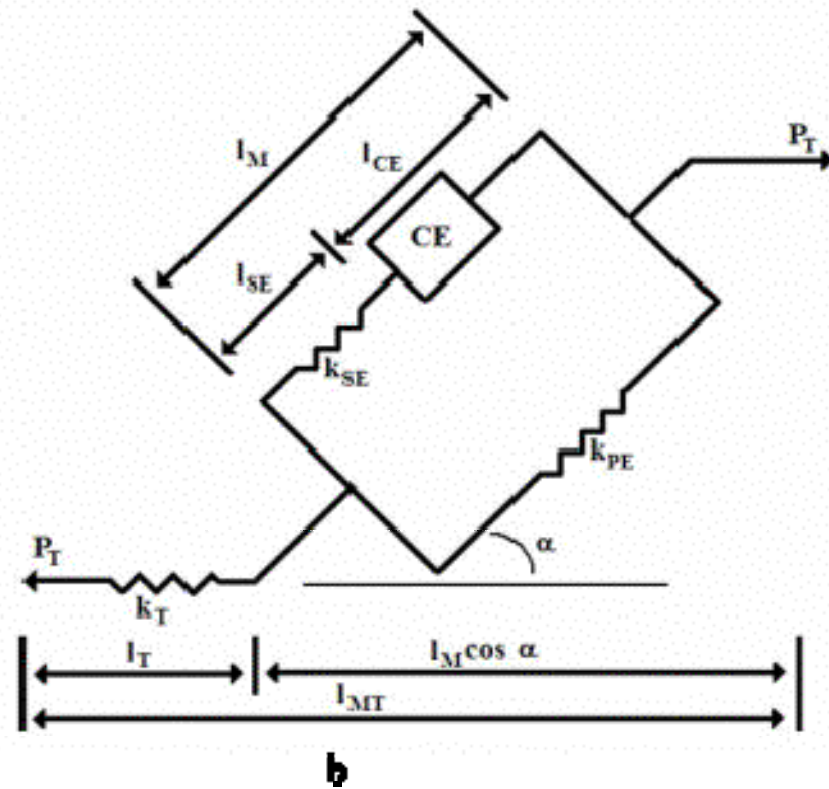
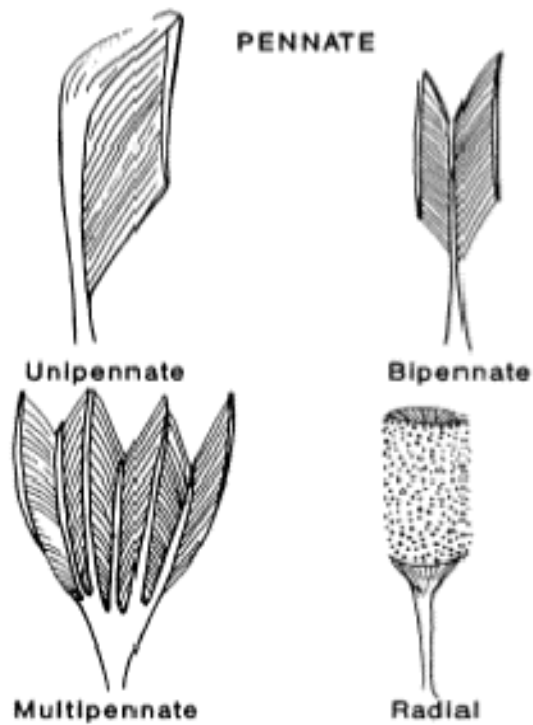
(2) Stress relaxation in muscle: $T_{ce}=0$; Passively stretch muscle

(3) Creep in muscles: $T_{ce}=0$; Adding a constant passive muscle external force

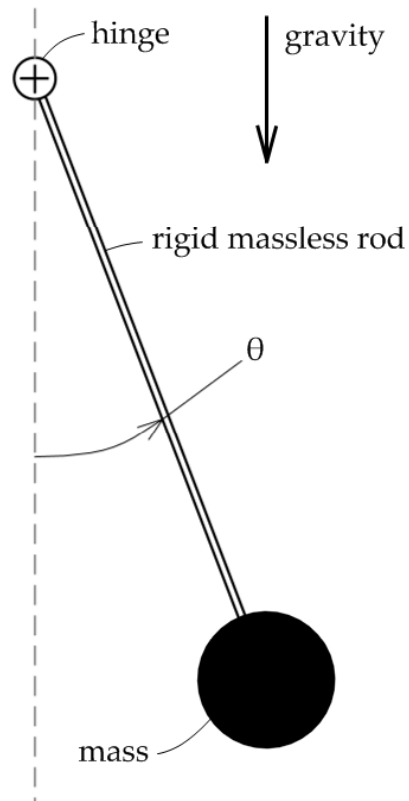
(4) Isotonic muscle contraction: T is a constant.

Improved Hill's Model

Zajac 1986



Nonlinear differential equations



$$ML \frac{d^2 \theta(t)}{dt^2} = Mg \sin \theta(t)$$



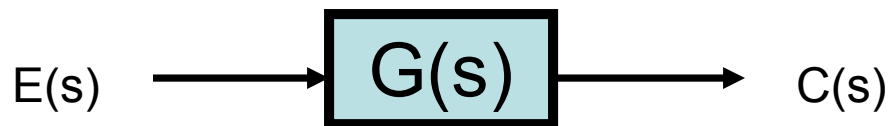
Small θ

$$ML \frac{d^2 \theta(t)}{dt^2} = Mg \theta(t)$$

<http://en.wikipedia.org/wiki/Nonlinearity>

Block Diagram

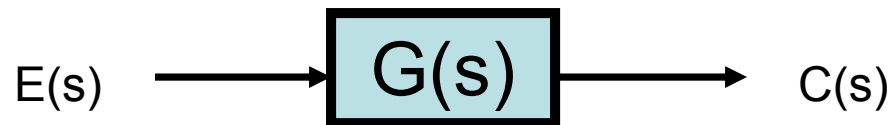
- $E(s)$: Laplace transform of input variable
- $C(s)$: Laplace transform of output variable
- $G(s)$: System transfer function



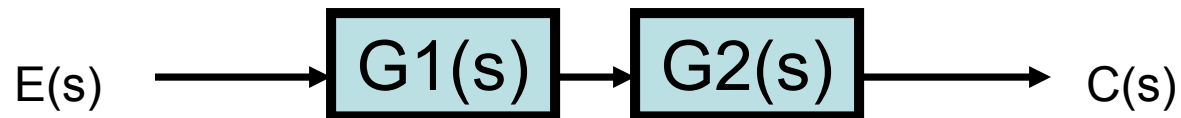
$C(s)=?$

Block Diagram

- The transfer function relationship $C(s)=G(s)E(s)$ can be graphically denoted through a block diagram



Series
connection

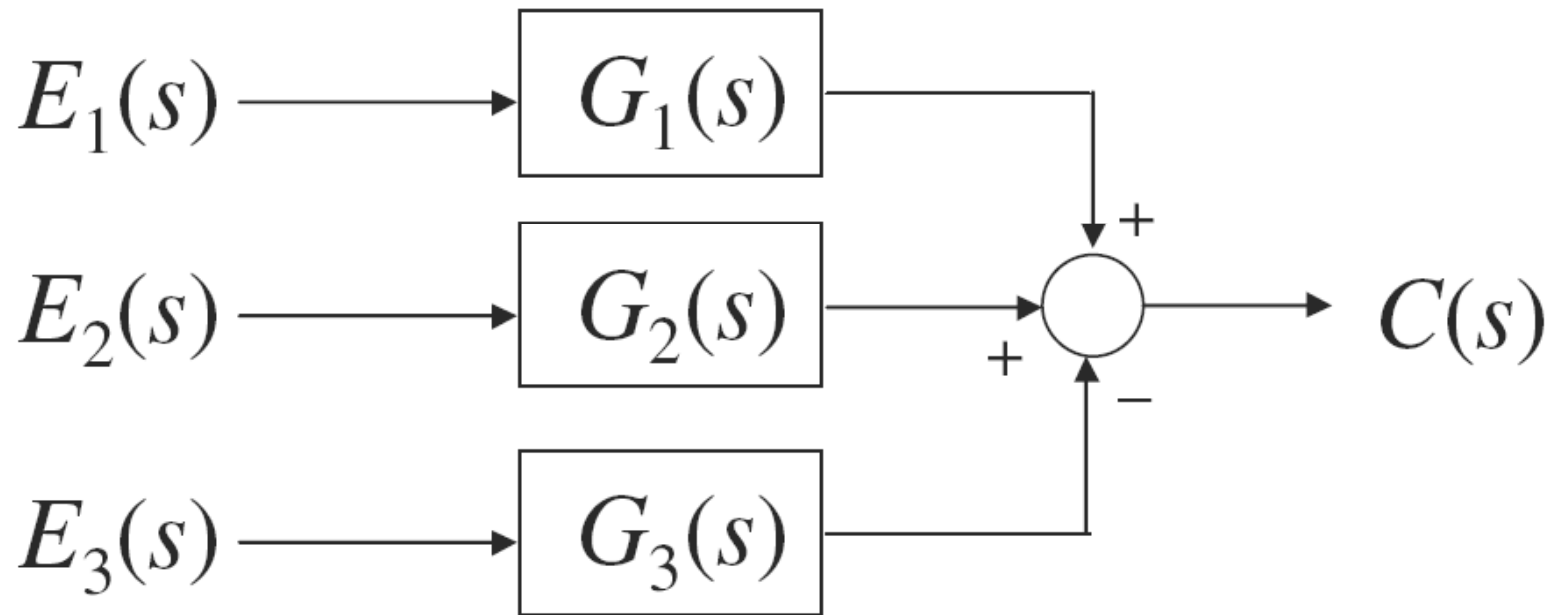


$$C(s) = G1(s)G2(s)E(s)$$

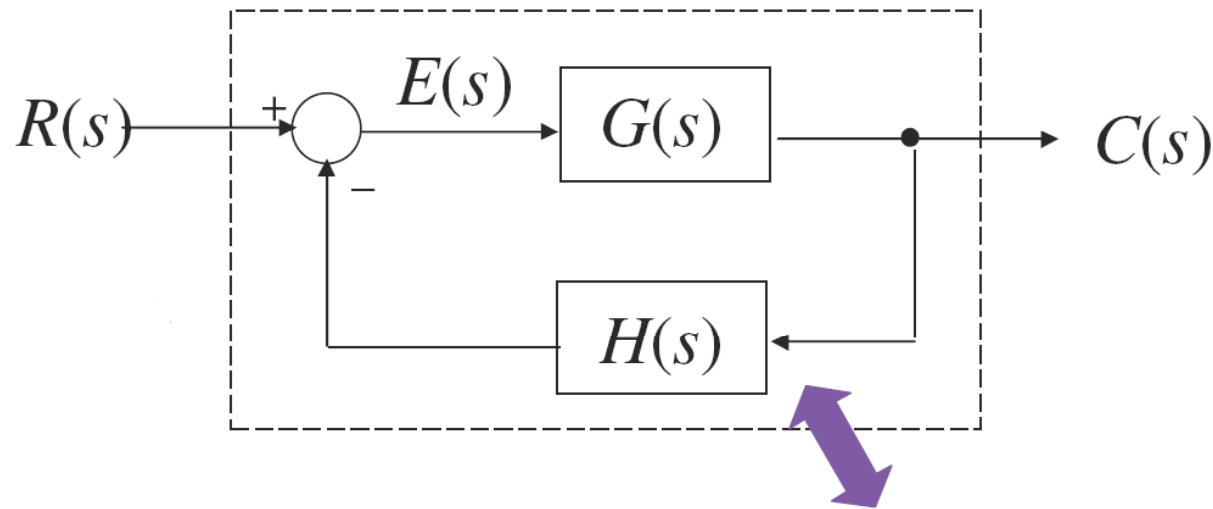
Block Diagram

$$C(s) = G_1(s)E_1(s) + G_2(s)E_2(s) - G_3(s)E_3(s)$$

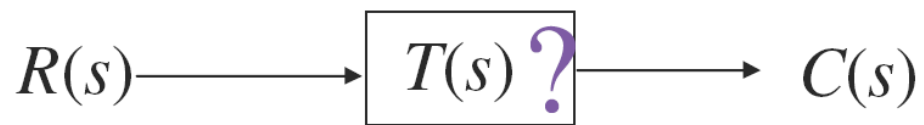
parallel connection



Block Diagram: Eliminating Variables



$$C(s) = E(s)G(s)$$



$$E(s) = \frac{C(s)}{G(s)} = R(s) - H(s)C(s)$$

$$T(s) = \frac{C(s)}{R(s)} = \frac{G(s)}{1 + G(s)H(s)}$$